## Strain



Challenger Explosion During Flight


- Relating strains to displacements is a problem in geometry.



## Learning objectives

- Learning the concept of strain.
- Learning the use of approximate deformed shape for calculating strains from displacements.


## Preliminary Definitions

- The total movement of a point with respect to a fixed reference coordinates is called displacement.
- The relative movement of a point with respect to another point on the body is called deformation.
- Lagrangian strain is computed from deformation by using the original undeformed geometry as the reference geometry.
- Eulerian strain is computed from deformation by using the final deformed geometry as the reference geometry.


## Average Normal Strain



$$
\varepsilon_{a v}=\frac{L_{f}-L_{o}}{L_{o}}=\frac{\delta}{L_{o}}
$$

- Elongations $\left(\mathrm{L}_{\mathrm{f}}>\mathrm{L}_{\mathrm{O}}\right)$ result in positive normal strains. Contractions $\left(\mathrm{L}_{\mathrm{f}}<\mathrm{L}_{\mathrm{o}}\right)$ result in negative normal strains.


$$
L_{0}=x_{B}-x_{A}
$$

$$
\varepsilon_{a v}=\frac{u_{B}-u_{A}}{x_{B}-x_{A}}
$$

$$
L_{f}=\left(x_{B}+u_{B}\right)-\left(x_{A}+u_{A}\right)
$$

$$
L_{f}=L_{o}+\left(u_{B}-u_{A}\right)
$$

## Units of average normal strain

- To differentiate average strain from strain at a point.
- in/in, or cm/cm, or m/m
- percentage.
$0.5 \%$ is equal to a strain of 0.005
- prefix: $\mu=10^{-6}$. $1000 \mu$ in / in is equal to a strain 0.001 in / in
2.5 Due to the application of the forces in Fig. P2.5, the displacement of the rigid plates in the x direction were observed as given below. Determine the axial strains in rods in sections $\mathrm{AB}, \mathrm{BC}$, and CD.

$$
u_{B}=-1.8 \mathrm{~mm} \quad u_{C}=0.7 \mathrm{~mm} \quad u_{D}=3.7 \mathrm{~mm}
$$



Fig. P2.5

## Average shear strain



Deformed grid


$$
\gamma_{a v}=\frac{\pi}{2}-\alpha
$$

- Decreases in the angle $(\alpha<\pi / 2)$ result in positive shear strain. Increase in the angle ( $\alpha>\pi / 2$ ) result in negative shear strain


## Units of average shear strain

- To differentiate average strain from strain at a point.
- rad
- prefix: $\mu=10^{-6}$. $1000 \mu \mathrm{rad}$ is equal to a strain 0.001 rad


| $\varepsilon_{\text {small }}$ Eq. 2.6 | $\varepsilon \quad$ Eq. 2.5 | $\%$ error |
| :---: | :---: | :---: |
| 1.0 | 1.23607 | 19.1 |
| 0.5 | 0.58114 | 14.0 |
| 0.1 | 0.10454 | 4.3 |
| 0.05 | 0.005119 | 2.32 |
| 0.01 | 0.01005 | 0.49 |
| 0.005 | 0.00501 | 0.25 |

- Small-strain approximation may be used for strains less than 0.01
- Small normal strains are calculated by using the deformation component in the original direction of the line element regardless of the orientation of the deformed line element.
- In small shear strain $(\gamma)$ calculations the following approximation may be used for the trigonometric functions: $\tan \gamma \approx \gamma \quad \sin \gamma \approx \gamma \quad \cos \gamma \approx 1$
- Small-strain calculations result in linear deformation analysis.
- Drawing approximate deformed shape is very important in analysis of small strains.
2.28 A thin triangular plate ABC forms a right angle at point A . During deformation, point A moves vertically down by $\delta_{\mathrm{A}}$. Determine the average shear strain at point A .

$$
\delta_{A}=0.008 \mathrm{in}
$$



Fig. P2.28
2.40 A roller at P slides in a slot as shown. Determine the deformation in bar AP and bar BP by using small strain approximation.


Fig. P2.40

## Class Problem 1

Draw an approximate exaggerated deformed shape.
Using small strain approximation write equations relating $\delta_{\mathrm{AP}}$ and $\delta_{\mathrm{BP}}$ to $\delta_{\mathrm{P}}$.



## Engineering Strain

Engineering strain matrix $\left[\begin{array}{lll}\varepsilon_{x x} & \gamma_{x y} & \gamma_{x z} \\ \gamma_{y x} & \varepsilon_{y y} & \gamma_{y z} \\ \gamma_{z x} & \gamma_{z y} & \varepsilon_{z z}\end{array}\right]$

Plane strain matrix

$$
\left[\begin{array}{ccc}
\varepsilon_{x x} & \gamma_{x y} & 0 \\
\gamma_{y x} & \varepsilon_{y y} & 0 \\
0 & 0 & 0
\end{array}\right]
$$

Strain at a point

$$
\begin{array}{ll}
\varepsilon_{x x}=\lim _{\Delta x \rightarrow 0}\left(\frac{\Delta u}{\Delta x}\right)=\frac{\partial u}{\partial x} \\
\gamma_{x y}=\gamma_{y x}=\lim _{\substack{\Delta x \rightarrow 0 \\
\Delta y \rightarrow 0}}\left(\frac{\Delta u}{\Delta y}+\frac{\Delta v}{\Delta x}\right)=\frac{\partial u}{\partial y}+\frac{\partial v}{\partial x} \\
\varepsilon_{y y}=\frac{\partial v}{\partial y} & \gamma_{y z}=\gamma_{z y}=\frac{\partial v}{\partial z}+\frac{\partial w}{\partial y} \\
\varepsilon_{z z}=\frac{\partial w}{\partial z} & \gamma_{z x}=\gamma_{x z}=\frac{\partial w}{\partial x}+\frac{\partial u}{\partial z}
\end{array}
$$

- tensor normal strains = engineering normal strains
- tensor shear strains $=($ engineering shear strains $) / 2$


## Strain at a Point on a Line

$$
\varepsilon_{x x}=\frac{d u(x)}{d x}
$$

2.54 Displacements $u$ and $v$ in the $x$ and $y$ directions respectively were measured by Moire Interferometry method at many points on a body. Displacements of four points on a body are given below. Determine the average values of strain components $\varepsilon_{x x}, \varepsilon_{y y}$, and $\gamma_{x y}$ at point A shown in Fig. P2.54.

|  | y |  | $u_{A}=0$ | $v_{A}=0$ |
| :---: | :---: | :---: | :---: | :---: |
|  |  |  | $u_{B}=0.625 \mu \mathrm{~mm}$ | $v_{B}=-0.3125 \mu \mathrm{~mm}$ |
|  | $\begin{array}{ll} \hline \mathrm{C} & \mathrm{D} \end{array}$ |  | $u_{C}=-0.500 \mu \mathrm{~mm}$ | $v_{C}=-0.5625 \mu \mathrm{~mm}$ |
|  | $\mathrm{A}$ $\mathrm{B}$ | $\xrightarrow{\mathrm{x}}$ | $u_{D}=0.250 \mu \mathrm{~mm}$ | $v_{D}=-1.125 \mu \mathrm{~mm}$ |

Fig. P2.54
2.60 The axial displacement in a quadratic one-dimensional finite element is as given below.

$$
u(x)=\frac{u_{1}}{2 a^{2}}(x-a)(x-2 a)-\frac{u_{2}}{a^{2}}(x)(x-2 a)+\frac{u_{3}}{2 a^{2}}(x)(x-a)
$$

Determine the strain at Node 2.

| Node 1 | Node 2 | Node 3 |
| :---: | :---: | :---: |
| - | $\bullet$ | $\bigcirc$ |
| $\mathrm{x}_{1}=0$ | $\mathrm{x}_{2}=\mathrm{a}$ | $\mathrm{x}_{3}=2 \mathrm{a}$ |

2.70 A metal strip is to be pulled and bent to conform to a rigid surface such that the length of strip between OA fits the arc OB of the surface. The equation of the surface $\mathrm{y}=\mathrm{f}(\mathrm{x})$ and the length OA is as given below. Determine the average normal strain in the metal strip. $f(x)=\left(0.04 x^{3 / 2}-0.005 x\right)$ inches and length $\mathrm{OA}=9$ inches. Use numerical integration


Fig. P2.70

