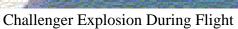
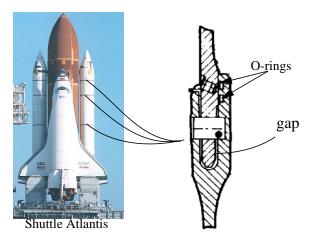
### **Strain**







• Relating strains to displacements is a problem in geometry.



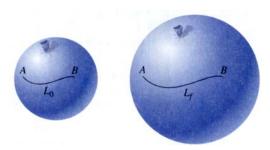
### **Learning objectives**

- Learning the concept of strain.
- Learning the use of approximate deformed shape for calculating strains from displacements.

## **Preliminary Definitions**

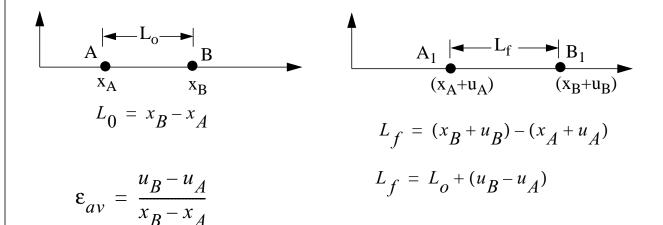
- The total movement of a point with respect to a fixed reference coordinates is called *displacement*.
- The relative movement of a point with respect to another point on the body is called *deformation*.
- Lagrangian strain is computed from deformation by using the original undeformed geometry as the reference geometry.
- *Eulerian strain* is computed from deformation by using the final deformed geometry as the reference geometry.

### **Average Normal Strain**



$$\varepsilon_{av} = \frac{L_f - L_o}{L_o} = \frac{\delta}{L_o}$$

• Elongations ( $L_f > L_o$ ) result in *positive* normal strains. Contractions ( $L_f < L_o$ ) result in *negative* normal strains.



#### Units of average normal strain

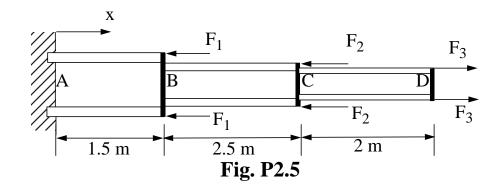
- To differentiate average strain from strain at a point.
- in/in, or cm/cm, or m/m
- percentage. 0.5% is equal to a strain of 0.005
- prefix:  $\mu = 10^{-6}$ . 1000  $\mu$  in / in is equal to a strain 0.001 in / in

2.5 Due to the application of the forces in Fig. P2.5, the displacement of the rigid plates in the x direction were observed as given below. Determine the axial strains in rods in sections AB, BC, and CD.

$$u_R = -1.8 mm$$

$$u_C = 0.7 mm$$

$$u_D = 3.7 \ mm$$



## Average shear strain

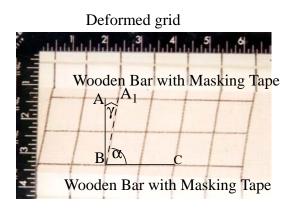
Undeformed grid

Wooden Bar with Masking Tape

B\pi/2

C

Wooden Bar with Masking Tape



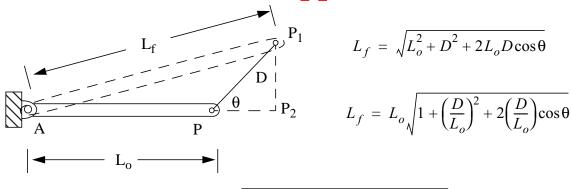
$$\gamma_{av} = \frac{\pi}{2} - \alpha$$

• Decreases in the angle  $(\alpha < \pi / 2)$  result in *positive* shear strain. Increase in the angle  $(\alpha > \pi / 2)$  result in *negative* shear strain

#### Units of average shear strain

- To differentiate average strain from strain at a point.
- rad
- prefix:  $\mu = 10^{-6}$ . 1000  $\mu$  rad is equal to a strain 0.001 rad

### **Small Strain Approximation**



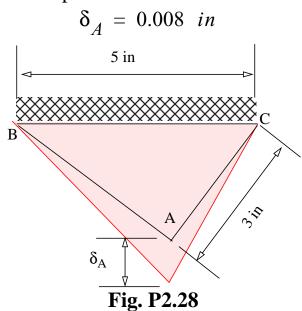
$$\varepsilon = \frac{L_f - L_o}{L_o} = \sqrt{1 + \left(\frac{D}{L_o}\right)^2 + 2\left(\frac{D}{L_o}\right)\cos\theta} - 1$$
 2.5

$$\varepsilon_{small} = \frac{D\cos\theta}{L_o}$$
 2.6

$\epsilon_{\rm small}$ Eq. 2.6	ε Eq. 2.5	% error
1.0	1.23607	19.1
0.5	0.58114	14.0
0.1	0.10454	4.3
0.05	0.005119	2.32
0.01	0.01005	0.49
0.005	0.00501	0.25

- Small-strain approximation may be used for strains less than 0.01
- Small normal strains are calculated by using the deformation component in the original direction of the line element regardless of the orientation of the deformed line element.
- In small shear strain ( $\gamma$ ) calculations the following approximation may be used for the trigonometric functions:  $\tan \gamma \approx \gamma \qquad \sin \gamma \approx \gamma \qquad \cos \gamma \approx 1$
- Small-strain calculations result in linear deformation analysis.
- Drawing approximate deformed shape is very important in analysis of small strains.

2.28 A thin triangular plate ABC forms a right angle at point A. During deformation, point A moves vertically down by  $\delta_A$ . Determine the average shear strain at point A.



2.40 A roller at P slides in a slot as shown. Determine the deformation in bar AP and bar BP by using small strain approximation.

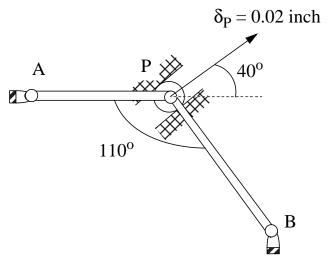
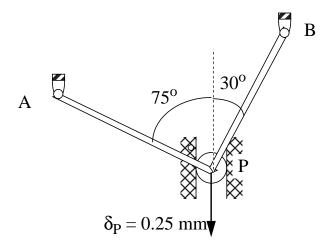


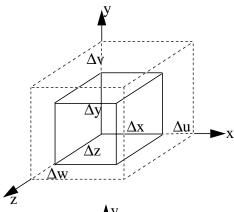
Fig. P2.40

# **Class Problem 1**

Draw an approximate exaggerated deformed shape. Using small strain approximation write equations relating  $~\delta_{AP}$  and  $~\delta_{BP}$  to  $\delta_{P.}$ 



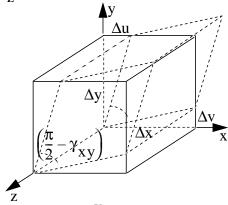
# **Strain Components**



$$\varepsilon_{xx} = \frac{\Delta u}{\Delta x}$$

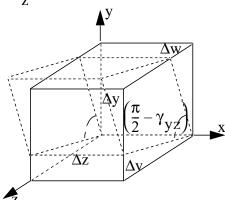
$$\varepsilon_{yy} = \frac{\Delta v}{\Delta y}$$

$$\varepsilon_{zz} = \frac{\Delta w}{\Delta z}$$



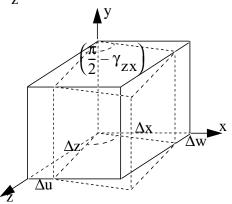
$$\gamma_{xy} = \frac{\Delta u}{\Delta y} + \frac{\Delta v}{\Delta x}$$

$$\gamma_{yx} = \frac{\Delta v}{\Delta x} + \frac{\Delta u}{\Delta y} = \gamma_{xy}$$



$$\gamma_{yz} = \frac{\Delta v}{\Delta z} + \frac{\Delta w}{\Delta y}$$

$$\gamma_{zy} = \frac{\Delta w}{\Delta y} + \frac{\Delta v}{\Delta z} = \gamma_{yz}$$



$$\gamma_{zx} = \frac{\Delta w}{\Delta x} + \frac{\Delta u}{\Delta z}$$

$$\gamma_{xz} = \frac{\Delta u}{\Delta z} + \frac{\Delta w}{\Delta x} = \gamma_{zx}$$

# **Engineering Strain**

**Engineering strain matrix** 

$$\begin{bmatrix} \varepsilon_{xx} & \gamma_{xy} & \gamma_{xz} \\ \gamma_{yx} & \varepsilon_{yy} & \gamma_{yz} \\ \gamma_{zx} & \gamma_{zy} & \varepsilon_{zz} \end{bmatrix}$$

Plane strain matrix

$$\begin{bmatrix} \varepsilon_{xx} & \gamma_{xy} & 0 \\ \gamma_{yx} & \varepsilon_{yy} & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

Strain at a point

$$\varepsilon_{xx} = \lim_{\Delta x \to 0} \left( \frac{\Delta u}{\Delta x} \right) = \frac{\partial u}{\partial x}$$

$$\gamma_{xy} = \gamma_{yx} = \lim_{\Delta x \to 0} \left( \frac{\Delta u}{\Delta y} + \frac{\Delta v}{\Delta x} \right) = \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x}$$

$$\Delta y \to 0$$

$$\varepsilon_{yy} = \frac{\partial v}{\partial y}$$

$$\gamma_{yz} = \gamma_{zy} = \frac{\partial v}{\partial z} + \frac{\partial w}{\partial y}$$

$$\varepsilon_{zz} = \frac{\partial w}{\partial z}$$

$$\gamma_{zx} = \gamma_{xz} = \frac{\partial w}{\partial x} + \frac{\partial u}{\partial z}$$

- tensor normal strains = engineering normal strains
- tensor shear strains = (engineering shear strains)/ 2

Strain at a Point on a Line

$$\varepsilon_{xx} = \frac{du(x)}{dx}$$

2.54 Displacements u and v in the x and y directions respectively were measured by Moire Interferometry method at many points on a body. Displacements of four points on a body are given below. Determine the average values of strain components  $\varepsilon_{xx}$ ,  $\varepsilon_{yy}$ , and  $\gamma_{xy}$  at point A shown in Fig. P2.54.

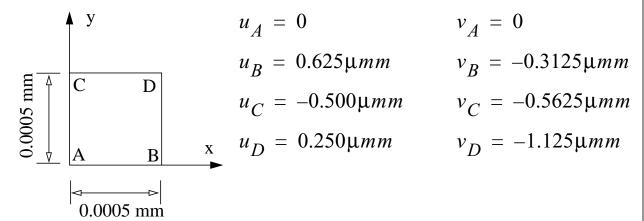
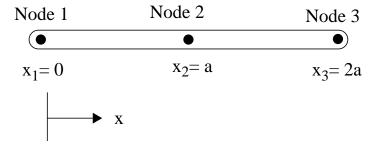


Fig. P2.54

2.60 The axial displacement in a quadratic one-dimensional finite element is as given below.

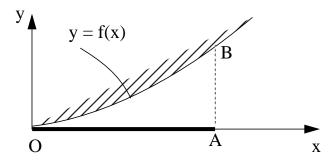
$$u(x) = \frac{u_1}{2a^2}(x-a)(x-2a) - \frac{u_2}{a^2}(x)(x-2a) + \frac{u_3}{2a^2}(x)(x-a)$$

Determine the strain at Node 2.



2.70 A metal strip is to be pulled and bent to conform to a rigid surface such that the length of strip between OA fits the arc OB of the surface. The equation of the surface y=f(x) and the length OA is as given below. Determine the average normal strain in the metal strip.

 $f(x) = (0.04x^{3/2} - 0.005x)$  inches and length OA = 9 inches. Use numerical integration



**Fig. P2.70**